## Math 3321

Vibrating Mechanical Systems

# University of Houston 

Lecture 12

## Outline

(1) Introduction
(2) Free Vibrations
(3) Undamped Free Vibrations

4 Damped Free Vibrations
(5) Forced Vibrations
(6) Undamped Forced Vibrations
(7) Damped Forced Vibrations

## Introduction

In this lecture, we will consider an application of second order linear differential equations.
Here we examine the motion of a spring of length $l_{0}$, from which an object of mass $m$ is attached. When the mass is attached, the spring extends to a length $l_{1}$. If the object is puled down or pushed up an additional $y_{0}$ units at time $t=0$ and then released, we are interested in the resulting motion of the object. We wish to find the position of the object at time $t>0$ ( $t$ measured in seconds).


## Introduction

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(A) Free Vibrations in which the forces acting on the spring-mass system are gravity and the restoring force of the spring. We will then include a damping force such as friction.
(B) Forced Vibrations in which an additional external force is applied to a freely vibrating system.

## Free Vibrations

We consider the forces acting on the object at time $t>0$. We begin with the downward force from gravity

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F_{1}=m g
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The second force is the restoring force from the spring. Hooke's Law tells us this force is proportional to the displacement $l_{1}+y(t)$ and acts in the direction opposite the displacement. That is:

$$
F_{2}=-k\left[l_{1}+y(t)\right] \text { where } k>0
$$

Here, $k$ is known as the spring constant.

## Undamped Free Vibrations

With undamped vibrations, we assume the forces $F_{1}$ and $F_{2}$ are the only forces. That is, we assume the spring is frictionless and there is no air resistence. Our force equation will be

$$
F=m g-k\left[l_{1}+y(t)\right]=\left(m g-k l_{1}\right)-k y(t)
$$

Before the object was displaced, the system was in equilibrium, so it must be

$$
m g-k l_{1}=0
$$

Therefore, the total force $F$ reduces to

$$
F=-k y(t)
$$

By Newton's Second Law of Motion, $F=m a$, hence

$$
m a=-k y(t) \quad \Rightarrow \quad a=y^{\prime \prime}(t)=-\frac{k}{m} y(t)
$$

## Undamped Free Vibrations

Since $\frac{k}{m}>0$, we introduce the notation $\omega^{2}=\frac{k}{m}$ and we write the last expression as

$$
y^{\prime \prime}(t)+\omega^{2} y(t)=0
$$

This is the equation of simple harmonic motion.

## Undamped Free Vibrations

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$$
y^{\prime \prime}(t)+\omega^{2} y(t)=0
$$

This is the equation of simple harmonic motion.
It is a second order linear homogeneous equation with constant coefficients whose characteristic equation is

$$
r^{2}+\omega^{2}=0
$$

The characteristic roots are $\pm i$, hence the general solution is

$$
y(t)=C_{1} \cos (\omega t)+C_{2} \sin (\omega t)
$$

## Undamped Free Vibrations

## Simple Harmonic Motion

Our equation of motion is

$$
\begin{equation*}
y^{\prime \prime}+\omega^{2} y=0 \tag{1}
\end{equation*}
$$

where $\omega=\sqrt{k / m}$. We found the solution to be

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where $\omega=\sqrt{k / m}$. We found the solution to be

$$
y=C_{1} \cos (\omega t)+C_{2} \sin (\omega t)
$$

We can rewrite this as

$$
y=A \sin (\omega t+\phi)
$$

where $A>0$ and $\phi \in[0,2 \pi)$. Here, the constant $\omega$ is the natural frequency of the system and $A$ is the amplitude of the motion. The number $\phi$ is the phase constant or phase shift.

## Undamped Free Vibrations

Below is a typical graph of a function $y=A \sin (\omega t+\phi)$


## Undamped Free Vibrations

Examples:

1. An object is in simple harmonic motion. Find an equation for the motion given that the period is $\frac{2 \pi}{3}$ and at time $t=0, y=1$, and $y^{\prime}=3$. What is the function which models this motion?

## Undamped Free Vibrations

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Since the period is $\frac{2 \pi}{\omega}$ and this quantity is equal to $\frac{2 \pi}{3}$, it follows that $\omega=3$ Hence the equation of motion is of the form

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To determine $A$ and $\phi$, we use the fact that

$$
1=y(0)=A \sin (\phi), \quad 3=y^{\prime}(0)=\left.3 A \cos (3 t+\phi)\right|_{t=0}=3 A \cos (\phi)
$$

which simplifies to

$$
A \sin (\phi)=1, \quad A \cos (\phi)=1
$$

## Undamped Free Vibrations

Thus we have that

$$
A^{2} \sin ^{2}(\phi)+A^{2} \cos ^{2}(\phi)=2 \quad \Rightarrow \quad A^{2}=2 \quad \Rightarrow \quad A=\sqrt{2}
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## Undamped Free Vibrations

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$$

Finally, to find $\phi$, we use the observation that

$$
\sqrt{2} \sin (\phi)=1, \sqrt{2} \cos (\phi)=1 \quad \Rightarrow \quad \sin (\phi)=\cos (\phi)=\frac{\sqrt{2}}{2}
$$

hence $\phi=\frac{\pi}{4}$. Thus, the equation of motion is

$$
y(t)=3 \sin (3 t+\pi / 4)
$$

## Undamped Free Vibrations

2. An object is in simple harmonic motion. Find an equation for the motion given that the frequency is $\frac{5}{\pi}$ and at time $t=0, y=1$, and $y^{\prime}=0$. What is the function which models this motion?

## Undamped Free Vibrations

2. An object is in simple harmonic motion. Find an equation for the motion given that the frequency is $\frac{5}{\pi}$ and at time $t=0, y=1$, and $y^{\prime}=0$. What is the function which models this motion?
Since the period is $\frac{2 \pi}{\omega}$ and this quantity is equal to $\frac{\pi}{5}=\frac{2 \pi}{10}$, it follows that $\omega=10$ Hence the equation of motion is of the form

$$
y(t)=A \sin (10 t+\phi)
$$

## Undamped Free Vibrations

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To determine $A$ and $\phi$, we use the fact that

$$
1=y(0)=A \sin (\phi), \quad 0=y^{\prime}(0)=\left.10 A \cos (10 t+\phi)\right|_{t=0}=10 A \cos (\phi)
$$

which simplifies to

$$
A \sin (\phi)=1, \quad \cos (\phi)=0
$$

Hence is must be $\phi=\pi / 2$ and $A=1$.

## Damped Free Vibrations

We now introduce a damping force $R$, such as friction or air resistence. A damping force resists the movement and experiments have shown that $R$ will be approximately proportional to the velocity $v=y^{\prime}$ and acts in the opposite direction relative to the motion. That is:

$$
R=-c y^{\prime} \text { with } c>0
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$$
R=-c y^{\prime} \text { with } c>0
$$

Our force equation is now

$$
F=-k y-c y^{\prime}
$$

and our equation of motion is

$$
\begin{equation*}
y^{\prime \prime}+\frac{c}{m} y^{\prime}+\frac{k}{m} y=0 \tag{2}
\end{equation*}
$$

This is the equation of motion in the presence of a damping factor.

## Damped Free Vibrations

The characteristic equation is

$$
r^{2}+\frac{c}{m} r+\frac{k}{m}=0
$$

and has roots

$$
r=\frac{-c \pm \sqrt{c^{2}-4 k m}}{2 m}
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$$

There are three cases to consider:

$$
c^{2}-4 k m<0, \quad c^{2}-4 k m>0, \quad c^{2}-4 k m=0
$$

## Damped Free Vibrations

## Case 1: $c^{2}-4 k m<0$

In this case, the characteristic equation has complex roots:

$$
r_{1}=-\frac{c}{2 m}+i \omega, r_{2}=-\frac{c}{2 m}-i \omega, \text { where } \omega=\frac{\sqrt{4 k m-c^{2}}}{2 m}
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## Damped Free Vibrations

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$$

Here the general solution is

$$
y=e^{(-c / 2 m) t}\left(C_{1} \cos (\omega t)+C_{2} \sin (\omega t)\right)
$$

which can also be written as

$$
\begin{equation*}
y=A e^{(-c / 2 m) t} \sin (\omega t+\phi) \tag{3}
\end{equation*}
$$

where $A>0$ and $\phi \in[0,2 \pi)$. We call this the underdamped case.

## Damped Free Vibrations

Below is a typical graph of a function (3).


## Damped Free Vibrations

## Case 2: $c^{2}-4 k m>0$

In this case, the characteristic equation has two distinct real roots:

$$
r_{1}=\frac{-c+\sqrt{c^{2}-4 k m}}{2 m}, r_{2}=\frac{-c-\sqrt{c^{2}-4 k m}}{2 m} .
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## Damped Free Vibrations

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$$

Here the general solution is

$$
\begin{equation*}
y=C_{1} e^{r_{1} t}+C_{2} e^{r_{2} t} . \tag{4}
\end{equation*}
$$

We call this the overdamped case.

## Damped Free Vibrations

## Case 3: $c^{2}-4 k m=0$

In this case, the characteristic equation has a single repeated root:

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r=-\frac{c}{2 m}
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Here the general solution is

$$
\begin{equation*}
y=C_{1} e^{-(c / 2 m) t}+C_{2} t e^{-(c / 2 m) t} \tag{5}
\end{equation*}
$$

We call this the critically damped case.

## Damped Free Vibrations

Below we see graphs which represent potential solutions in the overdamped and critically damped cases.



## Forced Vibrations

The motion we have studied thus far is the result of the interplay between three forces: gravity, the restoring force of the spring, and the resistence force of friction or the surrounding medium. These were free vibrations.

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We now apply an external force to our freely vibrating system and study the resulting motion. We call these forced vibrations.
Throughout, we will consider the application of a periodic external force of the form:

$$
F_{0} \cos (\gamma t)
$$

where $F_{0}$ and $\gamma$ are positive constants.

## Undamped Forced Vibrations

In an undamped system with an external force $F_{0} \cos (\gamma t)$, the behavior of our system will depend on the relationship between the applied frequency $\gamma$ and the natural frequency $\omega$. Here our force equation will be

$$
F=-k y+F_{0} \cos (\gamma t)
$$

Hence the equation of motion takes the form

$$
y^{\prime \prime}(t)+\frac{k}{m} y(t)=\frac{F_{0}}{m} \cos (\gamma t)
$$

By introducing the notation $\omega^{2}=\frac{k}{m}$ and above, we can write the last equation as

$$
y^{\prime \prime}(t)+\omega^{2} y(t)=\frac{F_{0}}{m} \cos (\gamma t)
$$

## Undamped Forced Vibrations

Let us examine the second order linear nonhomogeneous differential equation

$$
y^{\prime \prime}(t)+\omega^{2} y(t)=\frac{F_{0}}{m} \cos (\gamma t)
$$

By the properties of the particular solution of the differential equation, it is clear that the nature of the motion depends on the relation between the applied frequency $\gamma$ and the natural frequency $\omega$ of the system.

## Undamped Forced Vibrations

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y^{\prime \prime}(t)+\omega^{2} y(t)=\frac{F_{0}}{m} \cos (\gamma t)
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By the properties of the particular solution of the differential equation, it is clear that the nature of the motion depends on the relation between the applied frequency $\gamma$ and the natural frequency $\omega$ of the system.

Namely, if the $\omega \neq \gamma$ then the particular solution is of the form

$$
y_{p}=A \cos (\gamma t)+B \sin (\gamma t)
$$

However, if the $\omega=\gamma$ then the particular solution is of the form

$$
y_{p}=A t \cos (\gamma t)+B t \sin (\gamma t)
$$

## Undamped Forced Vibrations

## Case 1: $\gamma \neq \omega$

Here our equation of motion will be

$$
\begin{equation*}
y^{\prime \prime}+\omega^{2} y=F_{0} \cos (\gamma t) \tag{6}
\end{equation*}
$$

where $\omega=\sqrt{k / m}$. The method of undetermined coefficients will give a particular solution

$$
z=\frac{F_{0}}{m\left(\omega^{2}-\gamma^{2}\right)} \cos (\gamma t)
$$

and the general solution to our equation of motion is

$$
\begin{equation*}
y=A \sin (\omega t+\phi)+\frac{F_{0}}{m\left(\omega^{2}-\gamma^{2}\right)} \cos (\gamma t) \tag{7}
\end{equation*}
$$

## Undamped Forced Vibrations

Below we see a graph of a potential solution in the case where $\frac{\omega}{\gamma}$ is rational, in which case we see periodic motion.


## Undamped Forced Vibrations

Below we see a graph of a potential solution in the case where $\frac{\omega}{\gamma}$ is irrational, in which case we see motion which is not periodic.


## Undamped Forced Vibrations

Case 2: $\gamma=\omega$
Here our equation of motion will be

$$
y^{\prime \prime}+\omega^{2} y=\frac{F_{0}}{m} \cos (\omega t) .
$$

## Undamped Forced Vibrations

## Case 2: $\gamma=\omega$

Here our equation of motion will be

$$
y^{\prime \prime}+\omega^{2} y=\frac{F_{0}}{m} \cos (\omega t)
$$

The method of undetermined coefficients will give a particular solution

$$
z=\frac{F_{0}}{2 \omega m} t \sin (\omega t)
$$

and the general solution to our equation of motion is

$$
\begin{equation*}
y=A \sin (\omega t+\phi)+\frac{F_{0}}{2 \omega m} t \sin (\omega t) . \tag{8}
\end{equation*}
$$

The system is said to be in resonance. The motion will be oscillatory but the amplitude will grow linearly without bound.

## Undamped Forced Vibrations

Below we see a graph of a potential solution (8). We see oscillatory motion with an amplitude which grows linearly without bound.


## Damped Forced Vibrations

We introduce a damping force to our previous system of forced vibrations. Our force equation will be

$$
F=-k y-c y^{\prime}+F_{0} \cos (\gamma t)
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Hence the equation that governs the motion is

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m y^{\prime \prime}=-c y^{\prime}-k y+F_{0} \cos (\gamma t)
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Hence the equation that governs the motion is

$$
m y^{\prime \prime}=-c y^{\prime}-k y+F_{0} \cos (\gamma t)
$$

Using again the notation $\omega^{2}=k / m$, we can re-write it as

$$
y^{\prime \prime}+\frac{c}{m} y^{\prime}+\omega^{2} y=\frac{F_{0}}{m} \cos (\gamma t)
$$

## Damped Forced Vibrations

By the method of undetermined coefficients, we know that a particular solution of this equation will have the form

$$
y_{p}(t)=A \cos (\gamma t)+B \sin (\gamma t)
$$

which can be written as $y_{p}(t)=C \sin (\gamma t+\psi)$.

## Damped Forced Vibrations

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y_{p}(t)=A \cos (\gamma t)+B \sin (\gamma t)
$$

which can be written as $y_{p}(t)=C \sin (\gamma t+\psi)$.
Applying the method of undetermined coefficients, we obtain

$$
y_{p}(t)=\frac{F_{0}}{\sqrt{m^{2}\left(\omega^{2}-\gamma^{2}\right)^{2}+c^{2} \gamma^{2}}} \sin (\gamma t+\psi)
$$

## Damped Forced Vibrations

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Here our equation of motion will be

$$
\begin{equation*}
y^{\prime \prime}+\frac{c}{m} y^{\prime}+\frac{k}{m} y=\frac{F_{0}}{m} \cos (\gamma t) \tag{9}
\end{equation*}
$$

## Damped Forced Vibrations

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\end{equation*}
$$

The method of undetermined coefficients will give a particular solution which we use to build the general solution to our equation of motion

$$
\begin{equation*}
y(t)=y_{c}(t)+\frac{F_{0}}{\sqrt{m^{2}\left(\omega^{2}-\gamma^{2}\right)^{2}+c^{2} \gamma^{2}}} \sin (\gamma t+\psi) \tag{10}
\end{equation*}
$$

Here $y_{c}(t)$ is the general solution of the reduced equaton of (9) which we found in the damped free vibrations case. Recall, in all cases,

$$
\lim _{t \rightarrow \infty} y_{c}(t)=0
$$

## Damped Forced Vibrations

Example:

1. Give the solution of the IVP below.

$$
y^{\prime \prime}+y^{\prime}+\frac{101}{4} y=-200 \cos (2 t)-4 \sin (2 t), y(0)=4, y^{\prime}(0)=\frac{3}{2}
$$

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$$

The characteristic equation of the homogeneous equation is

$$
r^{2}+r+\frac{101}{2}=0
$$

and the roots are

$$
r=\frac{1- \pm 1-101}{2}=\frac{1}{2} \pm 5
$$

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The characteristic equation of the homogeneous equation is

$$
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$$

and the roots are

$$
r=\frac{1- \pm 1-101}{2}=\frac{1}{2} \pm 5
$$

Hence the general solution of the homogeneous equation is

$$
y_{h}(t)=C_{1} e^{-x / 2} \cos (5 t)+C_{2} e^{-x / 2} \sin (5 t)
$$

## Damped Forced Vibrations

The particular solution is of the form

$$
A \cos (2 t)+B \sin (2 t)
$$

When we apply the method of undetermined coefficients we find $A=2$ and $B=0$. Hence we get the general solution

$$
y=C_{1} e^{-x / 2} \cos (5 t)+C_{2} e^{-x / 2} \sin (5 t)+2 \cos (2 t)
$$

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$$
y=C_{1} e^{-x / 2} \cos (5 t)+C_{2} e^{-x / 2} \sin (5 t)+2 \cos (2 t)
$$

Imposing the initial conditions, we find $C_{1}=2, C_{2}=1 / 2$. Hence we have the IVP solution

$$
y=2 e^{-x / 2} \cos (5 t)+\frac{1}{2} e^{-x / 2} \sin (5 t)+2 \cos (2 t)
$$

## Damped Forced Vibrations

Below we see a graph of the transient solution

$$
y_{c}(t)=2 e^{-t / 2} \cos (5 t)+\frac{1}{2} e^{-t / 2} \sin (5 t)
$$



## Damped Forced Vibrations

Below we see a graph of the steady state solution

$$
z(t)=2 \cos (2 t)
$$



## Damped Forced Vibrations

Below we see a graph of the overall solution

$$
y(t)=y_{c}(t)+z(t)=2 e^{-t / 2} \cos (5 t)+\frac{1}{2} e^{-t / 2} \sin (5 t)+2 \cos (2 t)
$$



## Damped Forced Vibrations

Finally, we see the overall solution and the steady state solution graphed together.


