

Math 3331 Differential Equations

4.7 Forced Harmonic Motion

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4.7 Forced Harmonic Motion

- Periodically Forced Harmonic Motion
- Forced Undamped Harmonic Motion: Beats
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- Forced Damped Harmonic Motion
- Amplitude and Phase



Periodically Forced Harmonic Motion

Periodically forced mass-spring system: $mx'' + \mu x' + kx = F_0 \cos \omega t$

$$\text{or } x'' + dx' + \omega_0^2 x = A \cos \omega t$$

where $d = \mu/m$, $\omega_0 = \sqrt{k/m}$, $A = F_0/m$

Sinusoidal forcing: $F(t) = A \cos \omega t$

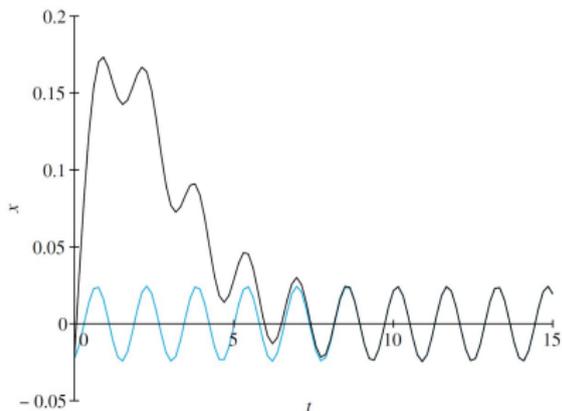
where A is the amplitude and ω is the driving frequency.

General solution:

$$x(t) = x_h(t) + x_p(t)$$

where

- $x_p(t)$: steady state part (persistent oscillation)
- $x_h(t)$: transient part ($d > 0$) ($x_h(t) \rightarrow 0$ for $t \rightarrow \infty$)



Forced Undamped Harmonic Motion: Beats ($\omega \neq \omega_0$)

$$x'' + \omega_0^2 x = A \cos \omega t \quad (1)$$

Try particular solution:

$$x_p(t) = a \cos \omega t + b \sin \omega t \Rightarrow$$

$$x_p'' + \omega_0^2 x_p = (\omega_0^2 - \omega^2)(a \cos \omega t + b \sin \omega t)$$

The r.h.s. is equal to $A \cos \omega t$ if

$$(\omega_0^2 - \omega^2)a = A, \quad (\omega_0^2 - \omega^2)b = 0$$

$$\Rightarrow a = A/(\omega_0^2 - \omega^2), \quad b = 0 \Rightarrow$$

$$x_p(t) = [A/(\omega_0^2 - \omega^2)] \cos \omega t \quad (2)$$

To find general solution, add general solution of

$$x'' + \omega_0^2 x = 0 \quad (3)$$

F.S.S. for (3): $\cos \omega_0 t, \sin \omega_0 t$

\Rightarrow general solution of (1):

$$x(t) = c_1 \cos \omega_0 t + c_2 \sin \omega_0 t + x_p(t)$$



Beats: $\omega \neq \omega_0$

Beats. Assume IC: $x(0) = 0, x'(0) = 0$

$$\Rightarrow c_1 = -A/(\omega_0^2 - \omega^2), c_2 = 0 \Rightarrow$$

$$x(t) = \frac{A}{\omega_0^2 - \omega^2}(\cos \omega t - \cos \omega_0 t) \quad (4)$$

Set $\delta = (\omega_0 - \omega)/2, \bar{\omega} = (\omega_0 + \omega)/2$

Use $(\alpha = \omega t, \beta = \omega_0 t)$

$$\cos \alpha - \cos \beta = 2 \sin\left(\frac{\beta - \alpha}{2}\right) \sin\left(\frac{\beta + \alpha}{2}\right)$$

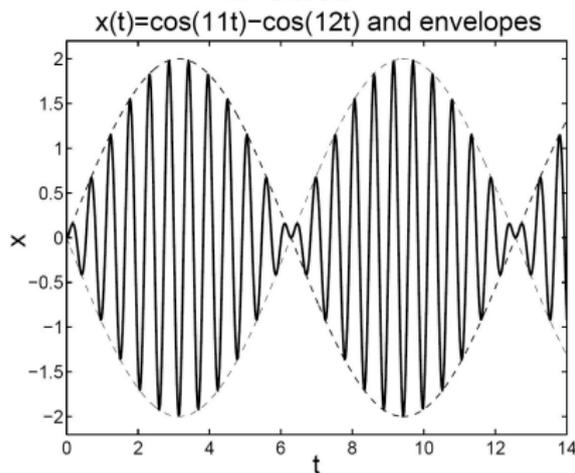
$$\Rightarrow x(t) = \frac{A \sin \delta t}{2\bar{\omega}\delta} \sin \bar{\omega} t \quad (5)$$

If $\delta \ll \bar{\omega} \Rightarrow [A/(2\bar{\omega}\delta)] \sin \delta t$ is slowly varying envelope



Beats in Forced, Undamped, Harmonic Motion

Beats



In acoustics, a beat is an interference between two sounds of slightly different frequencies, perceived as periodic variations in volume whose rate is the difference between the two frequencies.

$$x(t) = \cos \omega t - \cos \omega_0 t = 2 \sin \delta t \sin \bar{\omega} t$$

where the mean frequency $\bar{\omega}$ and the half difference δ are defined by

$$\bar{\omega} = (\omega_0 + \omega)/2, \quad \delta = (\omega_0 - \omega)/2.$$



Forced Undamped Harmonic Motion: Resonance ($\omega = \omega_0$)

Resonant Case: $\omega = \omega_0$

Solution (2) is not valid if $\omega = \omega_0$. In this case try

$$x_p(t) = t(a \cos \omega_0 t + b \sin \omega_0 t)$$

$$\begin{aligned} \Rightarrow x_p'' + \omega_0^2 x_p &= \\ &= -2a \sin \omega_0 t + 2b \cos \omega_0 t \end{aligned}$$

The r.h.s. equals $A \cos \omega_0 t$ if

$$a = 0, 2\omega_0 b = A \Rightarrow b = A/(2\omega_0)$$

$$\Rightarrow x_p(t) = [A/(2\omega_0)]t \sin \omega_0 t \quad (\text{linearly growing oscillation})$$

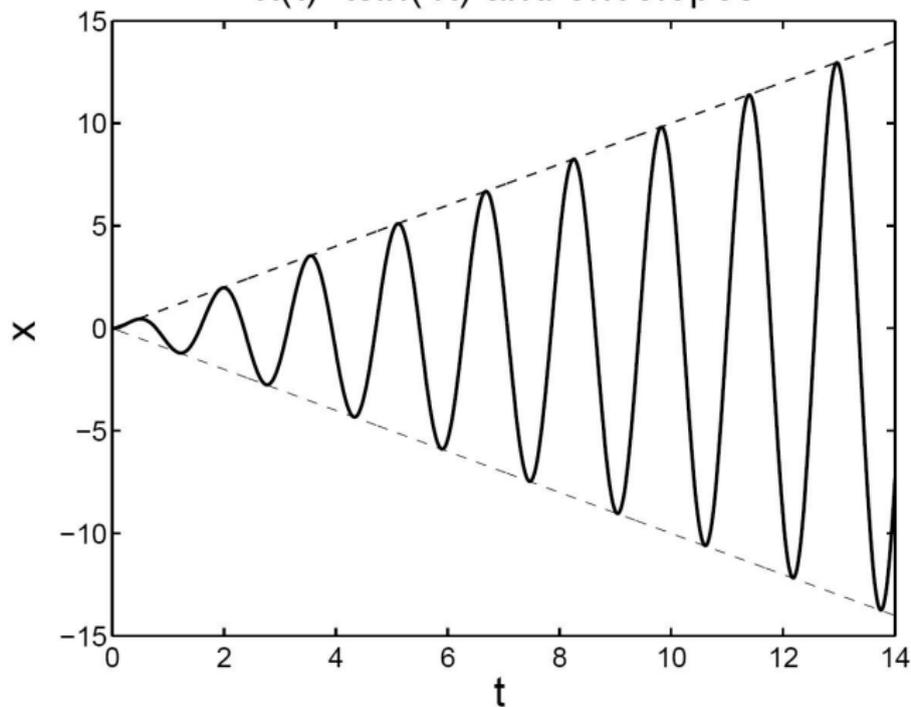
Note: $x_p(0) = 0, x_p'(0) = 0$. **Ex.:** $A = 8, \omega_0 = 4 \Rightarrow x_p(t) = t \sin 4t$



Resonance in Forced, Undamped, Harmonic Motion

Growing Oscillation

$x(t) = t \sin(4t)$ and envelopes



Forced Damped Harmonic Motion

$$x'' + dx' + \omega_0^2 x = A \cos \omega t \quad (6)$$

Since $A \cos \omega t = \operatorname{Re}(Ae^{i\omega t})$, any solution $x(t)$ is the real part of a solution $z(t)$ of

$$z'' + dz' + \omega_0^2 z = Ae^{i\omega t} \quad (7)$$

Solution Strategy:

- Find particular solution of (7)
- Real part \rightarrow particular solution of (6)



Particular Solution of (7)

Try **complex exponential** for (7):

$$z_p(t) = ae^{i\omega t} \Rightarrow z_p'' + dz_p' + \omega_0^2 z_p = ((i\omega)^2 + i\omega d + \omega_0^2)ae^{i\omega t} = Ae^{i\omega t}$$

$$\Rightarrow [(\omega_0^2 - \omega^2) + i\omega d]a = A$$

$$\Rightarrow \frac{a}{A} = \frac{1}{(\omega_0^2 - \omega^2) + i\omega d}$$

Use $1/(\alpha + i\beta) = (\alpha - i\beta)/(\alpha^2 + \beta^2)$

$$\Rightarrow \frac{a}{A} = \frac{(\omega_0^2 - \omega^2) - i\omega d}{D}$$

where $D = (\omega_0^2 - \omega^2)^2 + \omega^2 d^2$



Amplitude and Phase

Amplitude and Phase: Set

$$\begin{aligned}
 a/A &= Ge^{-i\phi} = G \cos \phi - iG \sin \phi \\
 \Rightarrow G^2 &= \left(\frac{(\omega_0^2 - \omega^2)}{D} \right)^2 + \left(\frac{\omega d}{D} \right)^2 \\
 &= \frac{(\omega_0^2 - \omega^2)^2 + \omega^2 d^2}{D^2} = \frac{D}{D^2}
 \end{aligned}$$

$$\Rightarrow G = 1/\sqrt{D} \equiv G(\omega) \text{ (gain), hence}$$

$$G(\omega) = \frac{1}{\sqrt{(\omega_0^2 - \omega^2)^2 + \omega^2 d^2}} \quad (8)$$

Phase angle:

$$\omega_0^2 - \omega^2 = G \cos \phi, \quad \omega d = G \sin \phi$$

where $0 \leq \phi < \pi$ (since $\sin \phi \geq 0$)

$$\Rightarrow \phi(\omega) = \operatorname{arccot} \left(\frac{\omega_0^2 - \omega^2}{\omega d} \right) \quad (9)$$



Solution of (6)

Particular Solution of (6):

$$z_p(t) = ae^{i\omega t} = G(\omega)Ae^{i(\omega t - \phi)} \Rightarrow$$

$$x_p(t) = \operatorname{Re}z_p(t) = GA \cos(\omega t - \phi) \quad (10)$$

General Solution of (6):

$$x(t) = x_h(t) + x_p(t) \quad (11)$$

$$\text{where } x_h(t) = c_1x_1(t) + c_2x_2(t) \quad (12)$$

and $x_1(t)$, $x_2(t)$ is F.S.S. of

$$x'' + dx' + \omega_0^2x = 0$$

Steady State and Transient Parts:

- $x_p(t)$: steady state part
(persistent oscillation)
- $x_h(t)$: transient part ($d > 0$)
 $\Rightarrow x_h(t) \rightarrow 0$ for $t \rightarrow \infty$)



Qualitative Forms

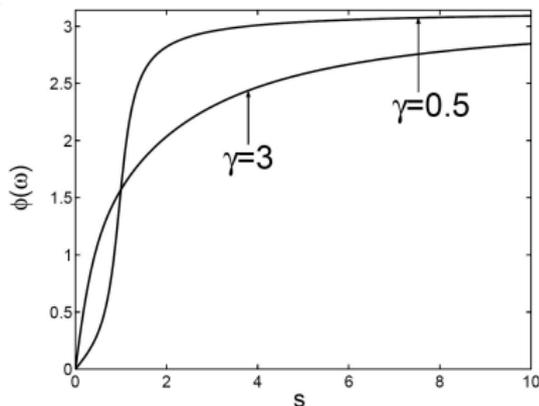
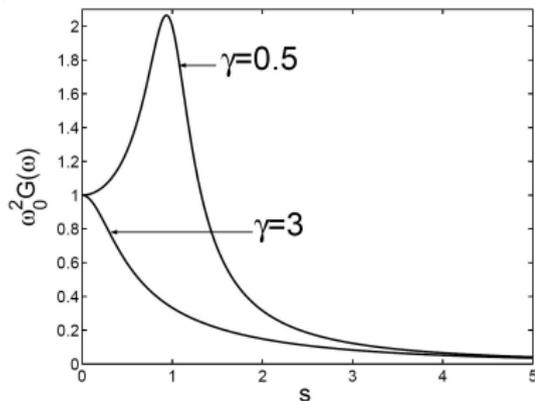
Qualitative Forms of $G(\omega)$, $\phi(\omega)$:

Set $s = \omega/\omega_0$, $\gamma = d/\omega_0 \Rightarrow$

$$\omega_0^2 G(\omega) = \frac{1}{\sqrt{(1-s^2)^2 + s^2\gamma^2}}$$

$$\phi(\omega) = \operatorname{arccot}\left(\frac{1-s^2}{s\gamma}\right)$$

- $G(\omega)$ has max at $s_m = \sqrt{1-\gamma^2/2}$, $\omega_0^2 G_m = 2/(\gamma\sqrt{4-\gamma^2})$, if $\gamma < \sqrt{2}$, and is monotonic for $\gamma > \sqrt{2}$
- $\phi(\omega)$ is "steep" for small γ and "flat" for large γ



Example 4.7.18

Ex.: Consider a mass-spring system with $m = 5 \text{ kg}$, $\mu = 7 \text{ kg/s}$, $k = 3 \text{ kg/s}^2$, and a forcing term $2 \cos 4t \text{ N}$

(a) Find the steady periodic solution $x_p(t)$ and determine its amplitude and phase.

Answer: Equation: $5x'' + 7x' + 3x = 2 \cos 4t \Rightarrow x'' + 1.4x' + 0.6x = 0.4 \cos 4t$

Use complex method: $x_p(t) = \text{Re}z_p(t)$, where z_p is particular solution of

$$z'' + 1.4z' + 0.6z = 0.4e^{4it}$$

Try $z_p = ae^{4it} \Rightarrow (-16 + 5.6i + 0.6)ae^{4it} = 0.4e^{4it}$

$$\Rightarrow a = \frac{0.4}{-15.4 + 5.6i} = \frac{0.4 \times (-15.4 - 5.6i)}{15.4^2 + 5.6^2} = -0.0229 - 0.0083i$$

$$\Rightarrow z_p(t) = (-0.0229 - 0.0083i)(\cos 4t + i \sin 4t)$$

$$\Rightarrow x_p(t) = \text{Re}(z_p(t)) = 0.0083 \sin 4t - 0.0229 \cos 4t \quad (\text{superposition form})$$

To find amplitude and phase compute polar form: $a = A_0 e^{-i\phi}$, where

$$A_0 = \sqrt{0.0229^2 + 0.0083^2} = 0.0244$$

$$\phi = \text{arccot}(-0.0229/0.0083) = 2.7939$$

$$\Rightarrow z_p(t) = A_0 e^{i(4t - \phi)}$$

$$\Rightarrow x_p(t) = 0.0244 \cos(4t - 2.7939) \quad (\text{amplitude-phase form})$$



Example 4.7.18 (cont.)

(b) Find the position $x(t)$ if $x(0) = 0$, $x'(0) = 1$ m/s

Answer: Find transient part: $x'' + 1.4x' + 0.6x = 0 \Rightarrow p(\lambda) = \lambda^2 + 1.4\lambda + 0.6 = 0$
 $\Rightarrow \lambda = -0.7 \pm 0.3317i$

$\Rightarrow x_h(t) = e^{-0.7t}[c_1 \cos(0.3317t) + c_2 \sin(0.3317t)]$ and $x(t) = x_h(t) + x_p(t)$

Match c_1, c_2 to IC: (use superposition form)

$$\left. \begin{aligned} x(0) &= c_1 - 0.0229 = 0 \Rightarrow c_1 = 0.0229 \\ x'(0) &= -0.7c_1 + 0.3317c_2 + 4 \times 0.0083 = 1 \Rightarrow c_2 = 2.9630 \end{aligned} \right\} \Rightarrow$$

$x(t) = e^{-0.7t}[0.0229 \cos(0.3317t) + 2.9630 \sin(0.3317t)] + 0.0244 \cos(4t - 2.7939)$

